The Control Volume Formulation for Non-Equilibrium Molecular Dynamics Simulations

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Overview

. What is a Control Volume (CV)?

. How do we apply it to Molecular Dynamics (MD)?

. How is it useful?



Overview

. What is a Control Volume (CV)?

• How do we apply it to Molecular Dynamics (MD)?

• How is it useful?















. The Control Volume is a purely conceptual closed surface used to analyse fluid flow



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. The Control Volume is a purely conceptual closed surface used to analyse fluid flow



- . What flows into a volume, minus what flows out
 - Mass conservation

$$\frac{\partial}{\partial t} \int_{V} \rho dV = \int_{in} \rho \boldsymbol{u} \cdot d\mathbf{S} - \int_{out} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

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- . What flows into a volume, minus what flows out
 - . Mass conservation

$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$



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- . What flows into a volume, minus what flows out
 - . Mass conservation

$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

Momentum Balance

$$\frac{\partial}{\partial t} \int_{V} \rho \boldsymbol{u} dV = -\oint_{S} \rho \boldsymbol{u} \boldsymbol{u} \cdot d\mathbf{S}$$

. The Control Volume is a purely conceptual closed surface used to analyse fluid flow



- . What flows into a volume, minus what flows out + Forces
 - . Mass conservation

$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

. Momentum Balance

$$\frac{\partial}{\partial t} \int_{V} \rho \boldsymbol{u} dV = -\oint_{S} \rho \boldsymbol{u} \boldsymbol{u} \cdot d\mathbf{S}$$

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 The Control Volume is a purely conceptual closed surface used to analyse fluid flow



- What flows into a volume, minus what flows out + Pressure
 - Mass conservation

$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

Momentum Balance

$$\frac{\partial}{\partial t} \int_{V} \rho \boldsymbol{u} dV = -\oint_{S} \rho \boldsymbol{u} \boldsymbol{u} \cdot d\mathbf{S}$$

 $\mathbf{\Pi} \cdot d\mathbf{S}$

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 The Control Volume is a purely conceptual closed surface used to analyse fluid flow



. What flows into a volume, minus what flows out + Pressure



The Control Volume Form

- The Control Volume is a purely conceptual closed surface used to analyse fluid flow
 - Can be defined anywhere in space with any shape
- . An alternate expression of the equations of motion
 - Mass conservation, momentum balance (Newton's law) and energy conservation
 - . Changes inside a volume exactly equal fluxes and forces over the surface
 - More fundamental¹ and general² as the continuum assumption is not required
- The MD system can be expressed in the same form, as a result:
 - Both continuum and discrete systems are expressed in the same manner
 - The surface fluxes and forces are the Method of Planes (MOP) form of stress
 - Nine MOP Stress can be obtained and exactly linked to the change of momentum inside the volume
 - Provides a unified framework for coupled simulations

^{1 -} Zienkiewicz The Finite Element Method for Fluid Dynamics 3rd edition

^{2 -} Kolditz (2001) -- Computional Methods in Environmental Fluid Mechanics C7 p132: integral formulations are required where discontinuous solutions are possible ... the integral formulation is the only physical meaningful problem description

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- . What flows into a volume, minus what flows out
 - . Mass conservation

$$\frac{d}{dt}\sum_{i=1}^{N}m_{i}\vartheta_{i} = -\sum_{i=1}^{N}m_{i}\mathbf{v}_{i} \cdot d\boldsymbol{S}_{i}$$

. The Control Volume is a purely conceptual closed surface used to analyse fluid flow



- What flows into a volume, minus what flows out + Forces
 - Mass conservation

$$\frac{d}{dt}\sum_{i=1}^{N}m_{i}\vartheta_{i} = -\sum_{i=1}^{N}m_{i}\mathbf{v}_{i} \cdot d\boldsymbol{S}_{i}$$

Momentum Balance

$$\frac{d}{dt} \sum_{i=1}^{N} m_i \mathbf{v}_i \vartheta_i = -\sum_{i=1}^{N} m_i \mathbf{v}_i \mathbf{v}_i \cdot d\mathbf{S}_i + \frac{1}{2} \sum_{i,j}^{N} \mathbf{f}_{ij} \vartheta_{ij}$$

 The Control Volume is a purely conceptual closed surface used to analyse fluid flow



What flows into a volume, minus what flows out + Pressure



Control Volume Function

• The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\vartheta_i \equiv \int_{x^-}^{x^+} \int_{y^-}^{y^+} \int_{z^-}^{z^+} \delta(x_i - x) \delta(y_i - y) \delta(z_i - z) dx dy dz$$

$$= [H(x^{+} - x_{i}) - H(x^{-} - x_{i})]$$

$$\times [H(y^{+} - y_{i}) - H(y^{-} - y_{i})]$$

$$\times [H(z^{+} - z_{i}) - H(z^{-} - z_{i})]$$



Derivatives yields the Surface Fluxes

. Taking the Derivative of the CV function

$$dS_{ix} \equiv -\frac{\partial \vartheta_i}{\partial x_i} = \left[\delta(x^+ - x_i) - \delta(x^- - x_i)\right] \\ \times \left[H(y^+ - y_i) - H(y^- - y_i)\right] \\ \times \left[H(z^+ - z_i) - H(z^- - z_i)\right]$$

• Surface fluxes over the top and bottom surface

$$dS_{ix} = dS_{ix}^+ - dS_{ix}^-$$

. Vector form defines six surfaces

$$d\mathbf{S}_i = \mathbf{i} dS_{xi} + \mathbf{j} dS_{yi} + \mathbf{k} dS_{zi}$$

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Control Volume Functional

• The Control volume function is the integral of the Dirac delta function in 3 dimensions



$$\vartheta_i \equiv \int_V \delta\left(\boldsymbol{r} - \boldsymbol{r}_i\right) dV$$

. Its derivative gives the fluxes over the surface



$$dS_{ix} \equiv -\frac{\partial \vartheta_i}{\partial x_i}$$

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Control Volume Functional - Forces

 A CV based on the length of intermolecular interaction inside the volume (used in the volume average stress)



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$$\vartheta_s \equiv \int_V \delta(\mathbf{r} - \mathbf{r}_i + s\mathbf{r}_{ij}) dV$$

 Its derivative gives the forces over the surface (as in the method of planes stress)



The Control Volume Equations

. Using the CV functional, the following equations are derived

Mass Conservation

$$\frac{d}{dt}\sum_{i=1}^{N}m_{i}\vartheta_{i} = -\sum_{i=1}^{N}m_{i}\mathbf{v}_{i} \cdot d\boldsymbol{S}_{i}$$

Momentum Balance



$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

$$\frac{\partial}{\partial t} \int_{V} \rho \boldsymbol{u} dV = -\oint_{S} \rho \boldsymbol{u} \boldsymbol{u} \cdot d\mathbf{S}$$

$$-\oint_{S} \mathbf{\Pi} \cdot d\mathbf{S}$$



Overview

• What is a Control Volume (CV)?

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. How is it useful?



Exact Conservation

. Results from any arbitrary volume

- Accumulation = Forcing + Advection
- Momentum evolution is the integral of accumulation



. The Method of Planes form of stress (Todd et al 1995)

 $\sigma = F/A$



. The Method of Planes form of stress (Todd et al 1995)



. The Method of Planes form of stress (Todd et al 1995)



- The Method of Planes form of stress (Todd et al 1995) appears naturally in the control volume equations
 - Different surfaces provide nine stress components (Cauchy stress theorem)
 - Localised MOP (Han and Lee 2004)



- . Non-uniqueness of the stress tensor is due to choice of volume
 - Stress is exactly linked to momentum change inside the volume

- . NEMD simulations often require applied constraints
 - . Typical constraints include barostats, thermostats, etc
 - . For coupled simulation, we need a "momentostat"
- . The constraint is localised using the control volume function
 - . The CV function takes care of the localisation for us
 - Momentum control in a CV requires the non-holonomic constraint

$$g(\boldsymbol{r}_i, \dot{\boldsymbol{r}}_i) = \sum_{i=1}^N m_i \dot{\boldsymbol{r}}_i \vartheta_i - \int_V \rho \boldsymbol{u} dV = 0$$

- . Gauss Principle of Least Constraint applied
 - · Valid for any form of constraint and provides physically meaningful trajectories
 - . CV function is mathematically well defined so we just work through the algebra

$$\frac{\partial}{\partial \boldsymbol{r}_{ij}} \sum_{i=1}^{N} \left[\boldsymbol{F}_{i} - \boldsymbol{r}_{ij} \right]^{2} - \boldsymbol{\lambda} \cdot \boldsymbol{g} = 0$$

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. Gauss Principle of Least Constraint applied

• Resulting constrained equations are differential e.g. the evolution of momentum is matched to a target momentum time evolution

$$m_i \ddot{\boldsymbol{r}}_i = \boldsymbol{F}_i + \frac{m_i \vartheta_i}{M_I} \left[\frac{d}{dt} \int_V \rho \boldsymbol{u} dV - \frac{d}{dt} \sum_{n=1}^N m_i \dot{\boldsymbol{r}}_i \vartheta_i \right]$$

 Surface fluxes and force exactly cancel the molecular terms and replace them with the target values

$$m_i \ddot{\boldsymbol{r}}_i = \boldsymbol{F}_i + \frac{m_i \vartheta_i}{M_I} \left[\frac{d}{dt} \int_V \rho \boldsymbol{u} dV + \sum_{n=1}^N m_i \dot{\boldsymbol{r}}_n \dot{\boldsymbol{r}}_n \cdot d\mathbf{S}_n - \sum_{n,m}^N \boldsymbol{f}_{nm} \mathbf{n} \cdot dS_{nm} \right]$$

- An iterative implementation of the constraint is required
 - Similar to SHAKE but iterating to cancel effects of momentum flux instead of bond length
 - Momentum control must be exact for a local differential constraint to be applied with no drift

Provides a method of controlling a volume's velocity and stress





- Zero time evolution applied
- No velocity evolution results
- Exact control of momentum using iteration to cancel both Forcing and Advection



Provides a method of controlling a volume's velocity and stress





- Cosinusoidal time evolution applied
- Sinusoidal velocity evolution results
- Exact control of momentum using iteration to cancel both Forcing and Advection


Constrained Control Volume

. Provides a method of controlling a volume's velocity and stress

$$\sum_{n,m}^{N} \boldsymbol{f}_{nm} \mathbf{n} \cdot d\mathbf{S}_{nm} = \sum_{n,m}^{N} \left[\boldsymbol{f}_{nm} dS_{xnm}^{+} + \boldsymbol{f}_{nm} dS_{xnm}^{-} + \boldsymbol{f}_{nm} dS_{ynm}^{+} + \boldsymbol{f}_{nm} dS_{ynm}^{-} \right]$$

• Forcing applies an arbitrary 18 component 3D stress field



Constrained Control Volume

Provides a method of controlling a volume's velocity and stress

$$\sum_{n,m}^{N} \boldsymbol{f}_{nm} \mathbf{n} \cdot d\mathbf{S}_{nm} = \sum_{n,m}^{N} \left[\boldsymbol{f}_{nm} dS_{xnm}^{+} + \boldsymbol{f}_{nm} dS_{xnm}^{-} + \boldsymbol{f}_{nm} dS_{ynm}^{+} + \boldsymbol{f}_{nm} dS_{ynm}^{-} \right]$$

. Forcing applies an arbitrary 18 component 3D stress field



Constrained Control Volume



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Couette Flow

. Applying Couette stresses and velocity evolution





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Coupling



Coupling Results – Couette Flow



Summary

. What is a Control Volume (CV)?

- Gives the equations of motion in integral form
- The only meaningful form for a discrete system
- . How do we apply it to Molecular Dynamics (MD)?
 - . Integrate the Irving and Kirkwood (1950) equations
 - You can then differentiate the CV functionals to get fluxes and forces

. How is it useful?

- Exact course graining of the MD equations (by selecting functionals)
- Providing nine Method of Planes stress components and exactly linking then to momentum evolution inside the CV
- Deriving Exact Constraints localised to a region in space
- Expressing MD in the same form as the CFD solver for coupled simulation

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Extra Material



Exact Conservation

. Mass Conservation



Exact Conservation

. Momentum Balance



Testing Momentum Balance

. Momentum Balance







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Testing Momentum Balance

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Testing Momentum Balance

. Momentum Balance



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Control Volume Function (revisited)

• The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\vartheta_i \equiv \int_V \delta(\mathbf{r} - \mathbf{r}_i) \, dV$$

= $\begin{bmatrix} H(x^+ - x_i) - H(x^- - x_i) \end{bmatrix}$
× $\begin{bmatrix} H(y^+ - y_i) - H(y^- - y_i) \end{bmatrix}$
× $\begin{bmatrix} H(z^+ - z_i) - H(z^- - z_i) \end{bmatrix}$

• Replace molecular position with equation for a line

$$m{r}_i
ightarrow m{r}_i - s m{r}_{ij}$$



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For full details, please see E.R. Smith, D.M. Heyes, D. Dini, T.A. Zaki, Phys. Rev. E 85. 056705 (2012)

Control Volume Function (revisited)

• The Control volume function is the integral of the Dirac delta function in 3 dimensions

$$\vartheta_s \equiv \int_V \delta(\mathbf{r} - \mathbf{r}_i + s\mathbf{r}_{ij})dV = \begin{bmatrix} H(x^+ - x_i + sx_{ij}) - H(x^- - x_i + sx_{ij}) \end{bmatrix} \\ \times \begin{bmatrix} H(y^+ - y_i + sy_{ij}) - H(y^- - y_i + sy_{ij}) \end{bmatrix} \\ \times \begin{bmatrix} H(z^+ - z_i + sz_{ij}) - H(z^- - z_i + sz_{ij}) \end{bmatrix}$$

. Length of interaction inside the CV

$$\ell_{ij} = \int_{0}^{1} \vartheta_s ds$$



Derivatives Yield the Surface Forces

. Taking the Derivative of the CV function

$$\frac{\partial \vartheta_s}{\partial x} \equiv \left[\delta(x^+ - x_i + sx_{ij}) - \delta(x^- - x_i + sx_{ij}) \right] \\ \times \left[H(y^+ - y_i + sy_{ij}) - H(y^- - y_i + sy_{ij}) \right] \\ \times \left[H(z^+ - z_i + sz_{ij}) - H(z^- - z_i + sz_{ij}) \right]$$

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• Surface fluxes over the top and bottom surface

$$dS_{xij} \equiv \int_{0}^{1} \frac{\partial \vartheta_s}{\partial x} ds = dS_{xij}^{+} - dS_{xij}^{-}$$

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$$dS_{xij}^{+} = \frac{1}{2} \underbrace{\left[sgn(x^{+} - x_{i}) - sgn(x^{+} - x_{j})\right]}_{MOP} S_{xij}$$

More on the Pressure Tensor

. Extensive literature on the form of the molecular stress tensor

- No unique solution Schofield, Henderson (1988)
- Two key forms in common use Volume Average (Lutsko, 1988) and Method of Planes (Todd et al 1995)
- . Link provided between these descriptions
 - Through formal manipulation of the functions
 - Exposes the relationship between the molecular stresses and the evolution of momentum
- In the limit the Dirac delta form of Irving and Kirkwood (1950) is obtained
 - This suggests the same limit is not possible in the molecular system
 - Arbitrary stress based on the volume of interest



Moving reference frame

- Why the continuum form of Reynolds' transport theorem has a partial derivative but the discrete is a full derivative
 - Eulerian mass conservation

$$\frac{d}{dt}\sum_{i=1}^{N} m_i \vartheta_i = -\sum_{i=1}^{N} m_i \mathbf{v}_i \cdot d\mathbf{S}_i$$

 $\vartheta_i = \vartheta_i(\boldsymbol{r}_i(t), \boldsymbol{r})$

$$\frac{\partial}{\partial t} \int_{V} \rho dV = -\oint_{S} \rho \boldsymbol{u} \cdot d\mathbf{S}$$

Lagrangian mass conservation

$$\frac{d}{dt}\sum_{i=1}^{N} m_i \vartheta_i = -\sum_{i=1}^{N} m_i \left(\mathbf{v}_i + \overline{\boldsymbol{u}}\right) \cdot d\boldsymbol{S}_i$$

 $\overline{\boldsymbol{u}} \cdot d\boldsymbol{S}_i = \frac{d\boldsymbol{r}}{dt} \cdot \frac{d\vartheta_i}{d\boldsymbol{r}}$

$$\vartheta_i = \vartheta_i(\boldsymbol{r}_i(t), \boldsymbol{r}(t))$$

$$\frac{d}{dt} \int_{V} \rho dV = \oint_{S} \rho \left(\boldsymbol{u} - \overline{\boldsymbol{u}} \right) \cdot d\boldsymbol{S}$$

$$\oint_{S} \rho \boldsymbol{u} \cdot d\boldsymbol{S} - \oint_{S} \rho \overline{\boldsymbol{u}} \cdot d\boldsymbol{S} = 0$$

Continuum Analytical Couette Flow



Where, $\lambda_n = \left(\frac{n\pi}{L}\right)^2$ and $u_n(t) = \frac{2U_0(-1)^n}{n\pi} \left(e^{-\frac{\lambda_n \mu t}{\rho}} - 1\right)$ **Imperial College**56

Unsteady Couette Flow

y

Continuum Analytical

• Simplify the momentum balance (Navier-Stokes) equation

$$\frac{\partial}{\partial t}\boldsymbol{u} + \boldsymbol{\nabla} \cdot \boldsymbol{u} \boldsymbol{u} = \frac{1}{\rho} \boldsymbol{\nabla} \boldsymbol{P} + \frac{\mu}{\rho} \boldsymbol{\nabla}^2 \boldsymbol{u}$$

• Solve the 1D unsteady diffusion equation.

$$\frac{\partial u_x}{\partial t} = \frac{\mu}{\rho} \frac{\partial^2 u_x}{\partial y^2}$$

• With Boundary Conditions

$$u_x(0,t) = 0$$
$$u_x(L,t) = U_0$$
$$u_x(y,0) = 0$$

• Molecular Dynamics

• Fixed bottom wall, sliding top wall with both thermostatted



Unsteady Couette Flow

Continuum Analytical

• Simplify the control volume momentum balance equation

$$\frac{\partial}{\partial t} \int_{V} \rho \boldsymbol{u} dV = -\oint_{S} \rho \boldsymbol{u} \boldsymbol{u} \cdot d\boldsymbol{S}$$
$$-\oint_{S} P \boldsymbol{I} \cdot d\boldsymbol{S} + \oint_{S} \boldsymbol{\sigma} \cdot d\boldsymbol{S}$$

. Simplifies for a single control volume

$$\frac{\partial}{\partial t}\int_{V}\!\!\!\!\rho u_{x}dV\!=\!\int_{S_{y}^{+}}\!\!\!\!\!\sigma_{xy}dS_{f}^{+}\!-\!\int_{S_{f}^{-}}\!\!\!\!\!\!\sigma_{xy}dS_{y}^{-}$$

• With Boundary Conditions

$$u_x(0,t) = 0$$
$$u_x(L,t) = U_0$$
$$u_x(y,0) = 0$$

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- Molecular Dynamics
 - Discrete form of the Momentum balance equation

$$\frac{d}{dt} \sum_{i=1}^{N} m_i \mathbf{v}_i \vartheta_i = -\oint_S \rho \boldsymbol{u} \boldsymbol{u} \cdot d\boldsymbol{S}$$
$$-\sum_{i=1}^{N} (\boldsymbol{v}_i - \boldsymbol{u}) (\boldsymbol{v}_i - \boldsymbol{u}) \cdot d\boldsymbol{S}_i - \sum_{i=1}^{N} \sum_{j \neq i}^{N} \varsigma_{ij} \cdot d\boldsymbol{S}_{ij}$$

• Simplifies for a single control volume

$$\frac{d}{dt}\sum_{i=1}^{N}m_i\mathbf{v}_i\vartheta_i = \sum_{i,j}^{N}f_{xij}dS_{yij}^+ - \sum_{i,j}^{N}f_{xij}dS_{yij}^-$$

• Fixed bottom wall, sliding top wall with both thermostatted



Unsteady Couette Flow

. Simulation setup

- . Starting Couette flow
- · Wall thermostat: Nosé-Hoover
- Averages are computed over 1000 time steps and 8 realizations





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Flow past a cylinder

. Use of the momentum conservation of the control volume to determine the drag coefficient



. Drag over a Carbon Nano-tube can be determined



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